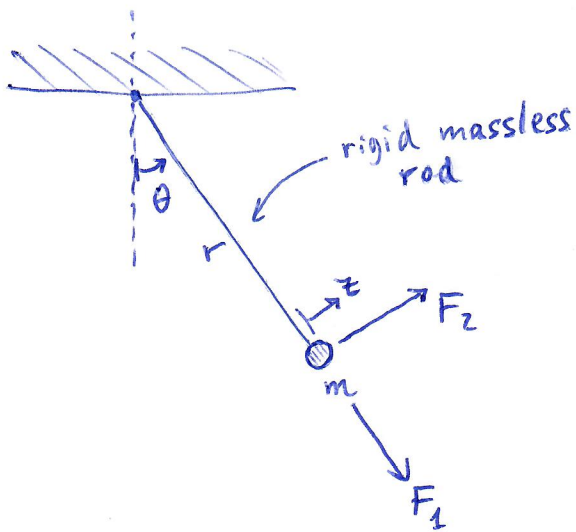


ME 4555 - Lecture 3 - Mechanical systems II

1

Today: mechanical systems with rotational motion
(rotation about an axis)

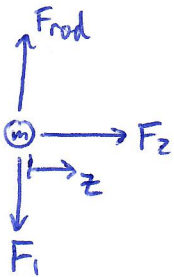
Newton's 2nd law for translational motion ($F = m\ddot{x}$) can be manipulated to characterize rotational motion as well.



F_1 (parallel to rod) is balanced by the rod, which prevents motion in that direction.*

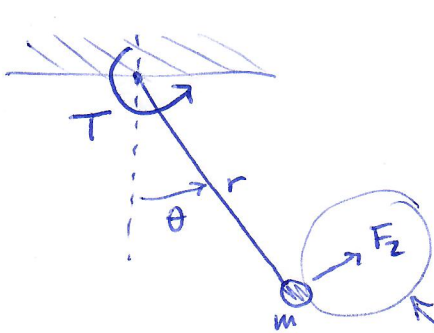
F_2 (perpendicular to rod) causes mass to move instantaneously in the direction of z .

FBD:



$$\left\{ \begin{array}{l} r\ddot{\theta} = \ddot{z} \\ m\ddot{z} = F_2 \end{array} \right\} \Rightarrow \boxed{mr\ddot{\theta} = F_2}$$

Such perpendicular force is often written as a Torque



$$\boxed{T = F_2 \cdot r}$$

so EOM in terms of θ, T :

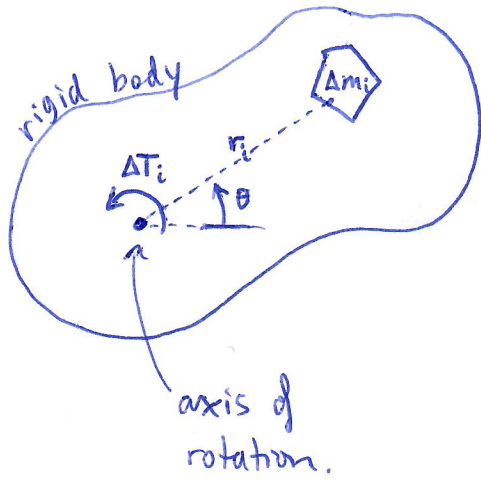
$$\boxed{mr^2\ddot{\theta} = T}$$

net result of torque T applied to hinge.

* It's more complicated than this, but let's ignore it for now since we only care about...

(2)

A rigid body rotating about an axis can be viewed as a collection of smaller masses, each rotating at the same angular rate about the axis.



From prev. page:

$$\Delta T_i = r_i^2 \Delta m_i \ddot{\theta}$$

Summing over all i (each mass piece):

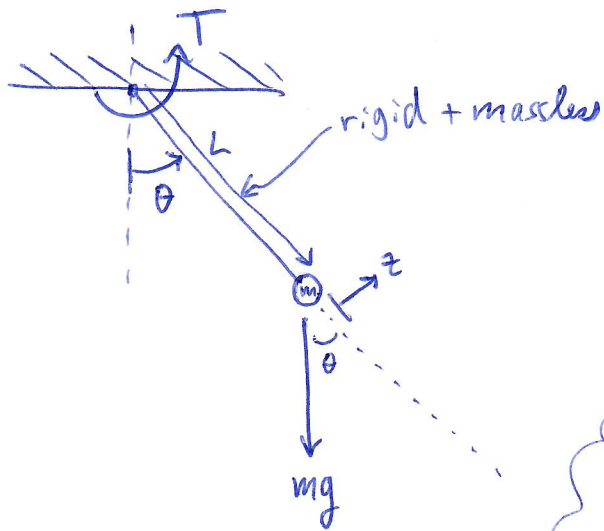
$$\underbrace{\sum_i \Delta T_i}_{\text{total torque at axis of rotation}} = \underbrace{\left(\sum_i r_i^2 \Delta m_i \right)}_{\text{moment of inertia (J)}} \underbrace{\ddot{\theta}}_{\text{angular acceleration}}$$

In the limit as Δm_i become infinitesimal, $J = \int r^2 dm$

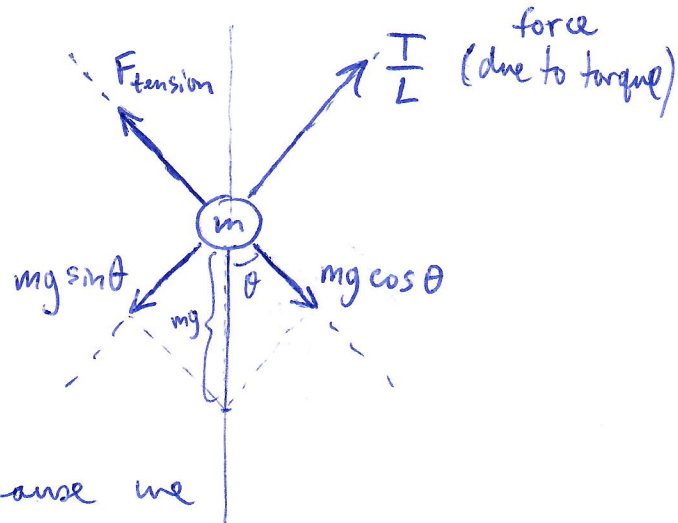
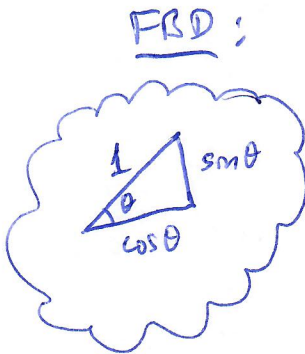
Newton's 2nd law for rotational motion:

$$\underbrace{\sum T}_{\text{sum of external torques}} = \underbrace{J}_{\text{moment of inertia of object about axis of rotation}} \underbrace{\ddot{\theta}}_{\text{angular acceleration}} \quad \left(\ddot{\theta} = \frac{d^2\theta}{dt^2} \right)$$

Ex: simple pendulum with gravity.



external forces decompose into parallel + perpendicular components.

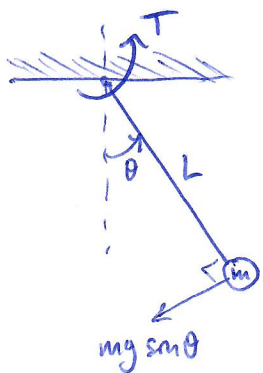


Let's ignore the parallel direction because we are only concerned with motion in the perpendicular direction.

In the perpendicular direction: $Lm\ddot{\theta} = -mg \sin \theta + \frac{T}{L}$

Rearranging: $\ddot{\theta} + \frac{g}{L} \sin \theta = \frac{T}{mL^2}$

different derivation: use rotational version!



for a mass m distance L from axis,

$J = mL^2$

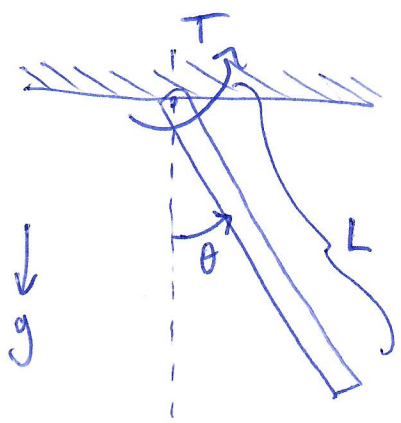
$J\ddot{\theta} = T - Lmg \sin \theta$

external applied torque torque due to gravity

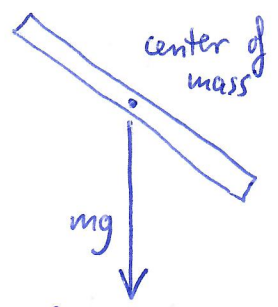
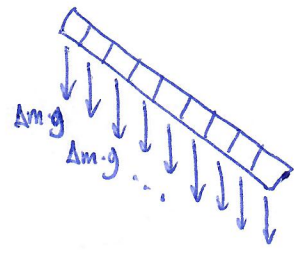
$\Rightarrow mL^2\ddot{\theta} + mgL \sin \theta = T$

same answer!

Ex: Solid pendulum



gravity acts on entire mass of arm



equivalent to putting all the force at the center of mass instead.

In other words: $\sum_i (\text{torque due to gravity on segment } i \text{ of the arm}) = (\text{distance to c.m.}) \cdot (\text{total gravity force})$

Assuming c.m. is $\frac{1}{2}$ way down ($\frac{L}{2}$ from hinge), we have:

$$J\ddot{\theta} = T - \left(\frac{1}{2}L\right)(mg \sin\theta)$$

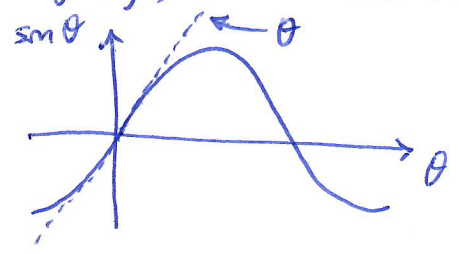
$$\Rightarrow \boxed{J\ddot{\theta} + \frac{1}{2}mgL \sin\theta = T}$$

This equation is nonlinear (depends on $\sin\theta$ instead of just θ).

If θ is close to zero (small angle), then use Taylor approximation

$$\sin\theta \approx \theta - \frac{\theta^3}{6} + \frac{\theta^5}{120} - \dots$$

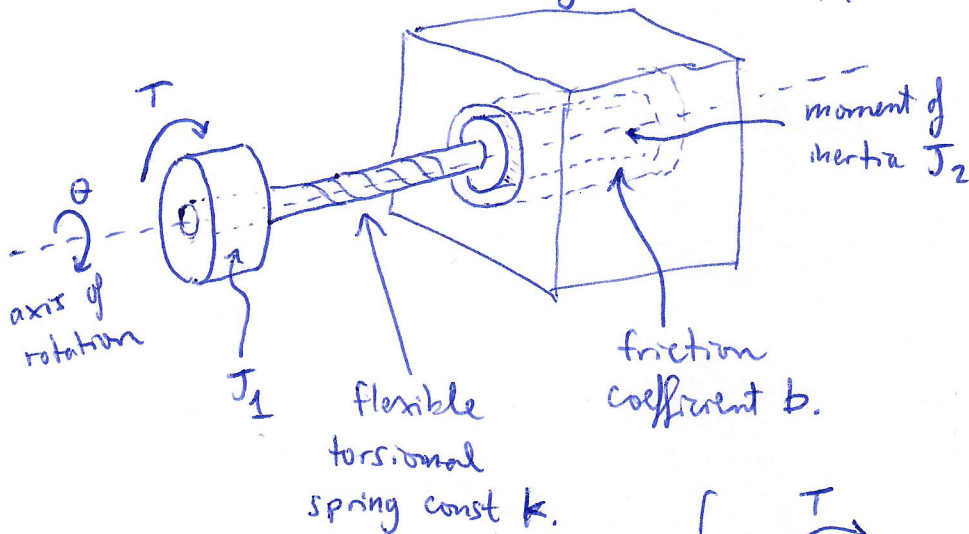
higher order terms



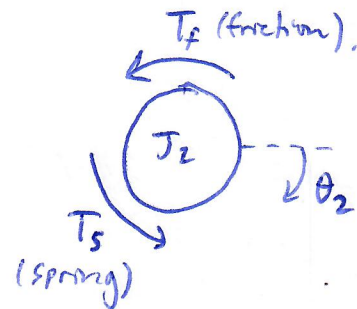
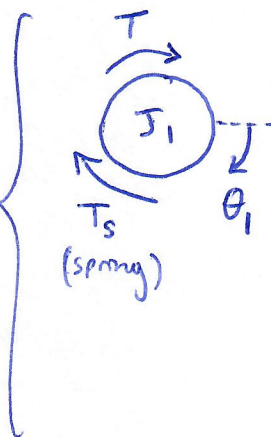
Then a linear approximation to the pendulum dynamics is:

$$\boxed{J\ddot{\theta} + \frac{1}{2}mgL \theta = T}$$

Ex: torsional spring-damper system



view from the left FBD:



First mass : $J_1 \ddot{\theta}_1 = T + \underbrace{k(\theta_2 - \theta_1)}_{T_s}$

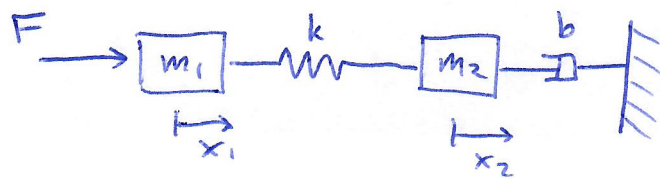
(spring torque caused by difference in angles)

Second mass : $J_2 \ddot{\theta}_2 = \underbrace{-k(\theta_2 - \theta_1)}_{T_s} - \underbrace{b\dot{\theta}_2}_{T_f}$

EOM solution :

$$\begin{cases} J_1 \ddot{\theta}_1 + k\theta_1 - k\theta_2 = T \\ J_2 \ddot{\theta}_2 + k\theta_2 - k\theta_1 + b\dot{\theta}_2 = 0 \end{cases}$$

Analogous linear translational system:



if we let $m_i \leftrightarrow J_i$
 $F \leftrightarrow T$
 $x_i \leftrightarrow \theta_i$
 the EOM are the same!